


1

AN EMPIRICAL MODEL FOR SHIP-GENERATED WAVES



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2

Objectives Ship-Generated Wave Heights

- Develop empirical equation to predict maximum wave heights generated by conventional ships
- Improve upon existing predictive equations:
 - Sorensen and Weggel (1984) & Weggel and Sorensen (1986)
 - PIANC (1987)
- Use existing data published in the literature
- Run new lab tests to supplement existing data
- Run field trials to verify empirical model
 - Tests conducted in Chesapeake Bay April 2005

Definitions

- **Two wave systems:**
 - Diverging Waves
 - Transverse Waves
- **Maximum wave heights**
 - Form along “Cusp Line”
 - Vary with distance from sailing line, y

SAMPLE SHIP-GENERATED WAVE PATTERN
FOR DEEP WATER
(after Sorensen, 1997)

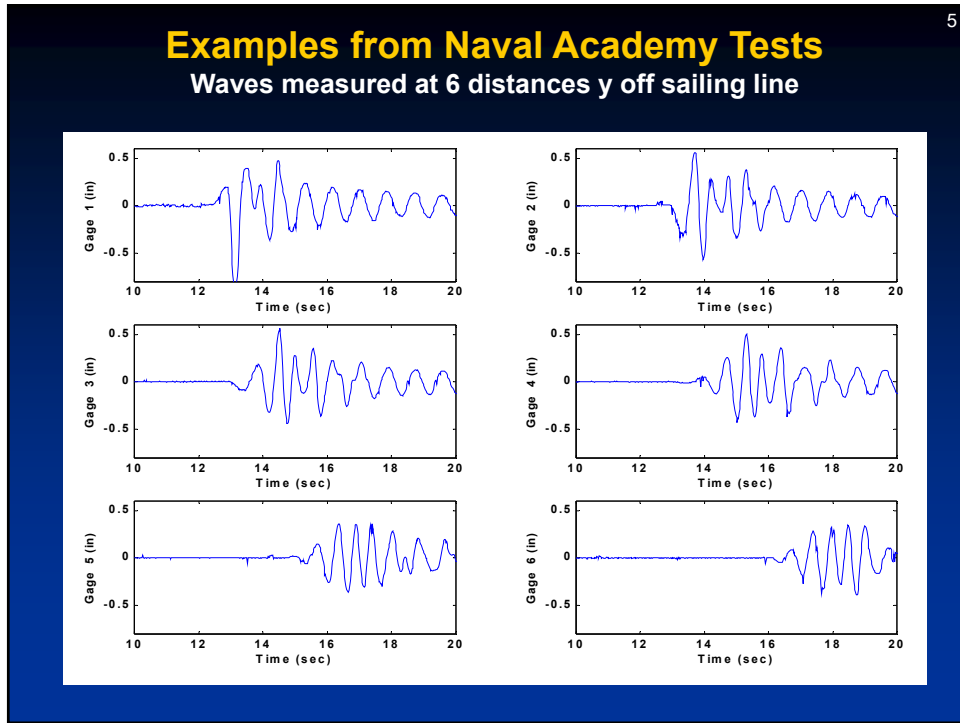
Sample Wave Records from Literature

Deep Water
from Das (1969)

MARINER MODEL
SCALE 1:96
LENGTH = 5.9 FT
DEPTH OF WATER = 25 IN = 2.08 FT
SPEED OF SHIP = 3.8 FT/SEC
 $Fr_h = 0.46$

Shallow Water
from Das (1969)

MARINER MODEL
SCALE 1:96
LENGTH = 5.9 FT
DEPTH OF WATER = 4 1/2 IN = 0.344 FT
SPEED OF SHIP = 2.7 FT/SEC
 $Fr_h = 0.81$



Ship Wave Database

Ship Number	Investigator	Scale for R=10 m	Vessel	Scale	Displacement (lbs)	Length (feet)	Lwt (feet)	Beam (feet)	Draft (feet)	
1	Sorensen (1973)	8.73	Cabin Cruiser	full	6,000	23	8.3	1.70		
2		8.58	Coast Guard Cutter (40-FOOT)	full	19,730	40	37	10.7	1.91	
3		3.71	Tugboat	full	58,000	45	12	6.00		
4		5.29	Air-Sea Rescue Vessel							
5		1.87	Fireboat							
6		1.18	Barge							
7		0.77	Moore Dry Dock Tanker							
8	Sorensen (1966)	185.59	Model A							
9		139.12	Model B							
10		111.40	Model C							
11		92.80	Model D							
12		69.61	Model E							
13		77.83	Weinblum hull							
14	Hay (1967)	76.11	Mariner Class Cargo Ship							
15		81.58	SERIES 60 Cb=0.6							
16		74.05	Moore Dry Dock Tanker							
17		58.20	Auxiliary Supply Vessel							
18		57.02	Barge							
19		47.88	Tug							
20	Bkide (1968)	76.11	Mariner Class Cargo Ship (A)							
21		76.11	Barge (E)							
22	Das (1969)	76.11	Mariner Class Cargo Ship							
23			Cruiser							
24	Zabawa & Ostroa (1980)		Uniflight Cruiser							
25			Boston Whaler							
26	Kurata & Oda (1984)	64.76	Ferryboat							
27			Tugboat							
28	USNA (2000)	78.58	SERIES 60 Cb=0.6 MID							
29	USNA (2000)	75.40	SERIES 60 Cb=0.6 HEAVY							
30	USNA (2000)	83.10	SERIES 60 Cb=0.6 LIGHT							
41	Helwig (1966)		Empress of Canada (Ocean Liner)							
42	Helwig (1966)		M.S. Wearfield (Ocean Freighter)							
47	Carruthers (1966)	73.64	Cape Breton Miner Bulk Carrier No Bulb	1:57.8	347,992	11,763	11,180	58	1,245	0.448745
48	Carruthers (1966)	44.28	Cape Breton Miner Bulk Carrier No Bulb	1:57.8	191,718	7,021	6,489	26	1,245	0.448745
49	Carruthers (1966)	44.28	Cape Breton Miner Bulk Carrier No Bulb SHORT	1:56	81	7,083	6.72	0.75	0.286	
50	Helwig (1966)	71.56	Cape Breton Miner Bulk Carrier Bulb	1:96	81	7,083	6.72	0.75	0.286	
51	Helwig (1966)	71.56	Cape Breton Miner Bulk Carrier No Bulb	1:96	81	7,083	6.72	0.75	0.286	

Ship wave data used in present analysis:
2100 data points for 12 ships

Sample Wave Data from Literature

- Wave heights for a given ship hull form characterized by:
 - Length, L
 - Beam, B
 - Draft, D
 - Hull Form
- Wave Heights vary with:
 - Ship speed, V
 - Distance from sailing line, y
 - Water depth, d

(FROM HELWIG, 1966)

Normalizing Wave Measurements

- Wave height normalized by velocity head

$$gH/V^2$$
- Distance from sailing line can be normalized in many ways (with various length scales L, B, etc) ...we use:

$$y/L$$
- Data in shallow water organized by depth-to-draft ratio

$$D/d$$
- Velocities normalized as Froude Number

$$F_d = V/(gd)^{1/2} \quad \text{or} \quad F_L = V/(gL)^{1/2}$$

9

Empirical Model

Variation of H with Distance from Sailing Line

- Havelock (1908) theory for deep water:
 - $H \sim y^{-1/3}$ for diverging waves
 - $H \sim y^{-1/2}$ for transverse waves
- Empirical evidence in literature shows a range from
 - $H \sim y^{-0.25}$ to $H \sim y^{-0.6}$
- Present study:
 - Least-squares fit of both $-1/3$ and $-1/2$ models to data
 - Model using $-1/3$ power gave best fit
 - C varies with ship hull form, T/D , F_d or F_L

$$\frac{gH}{V^2} = C \left(\frac{y}{L} \right)^{-1/3}$$

Examples of Fit

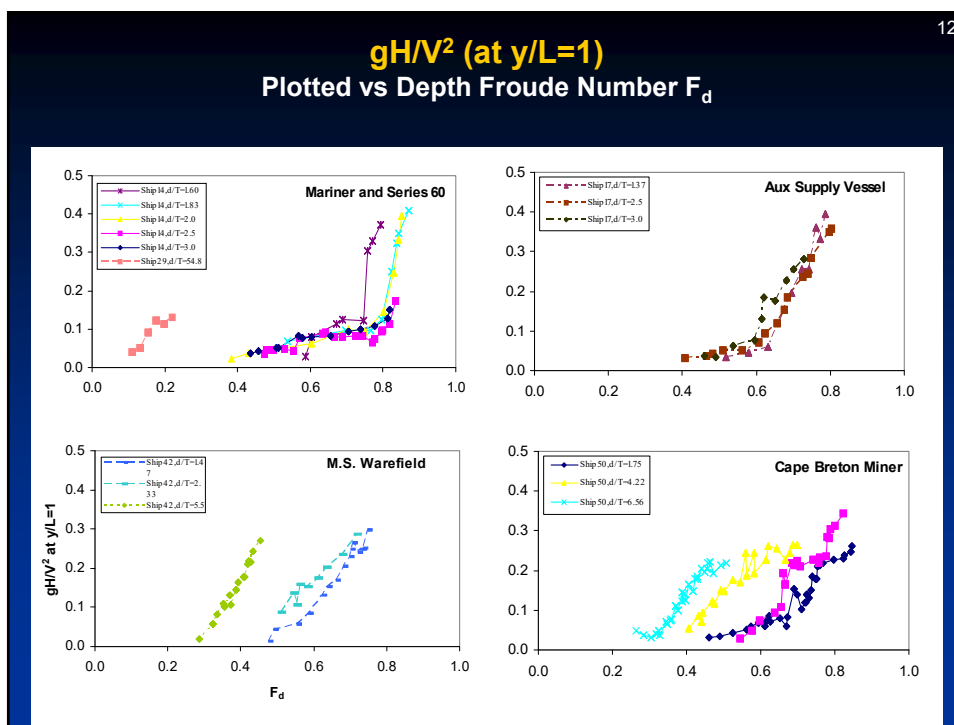
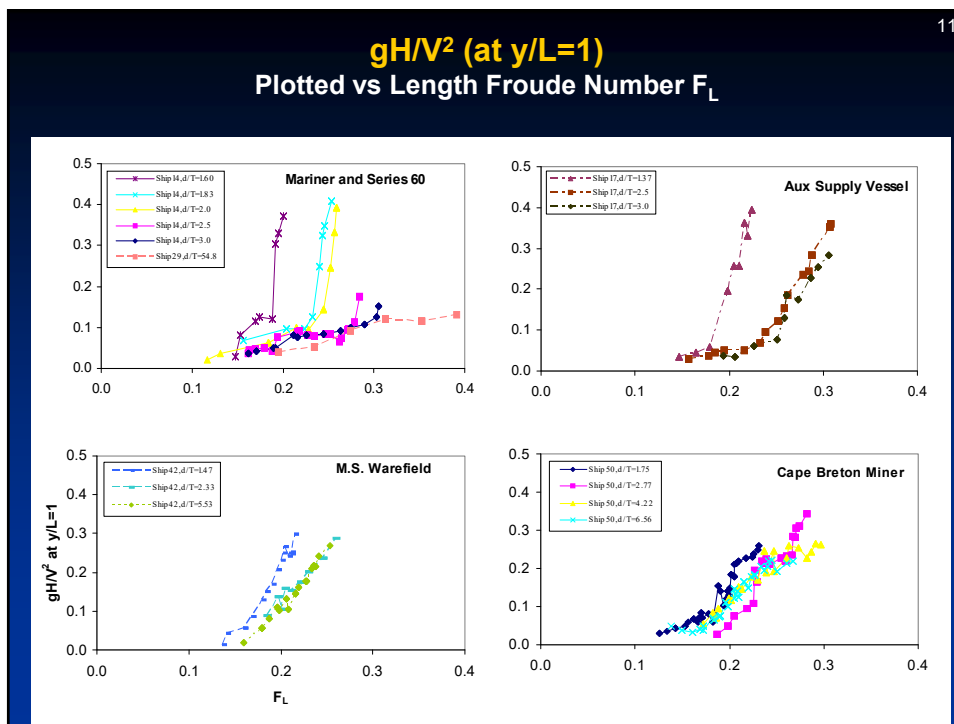
gH/V^2 versus $(y/L)^{-1/3}$

Value of gH/V^2 at $y/L=1$ used as a characteristic value for further analysis

Nondimensional H_{max} versus Distance from the Sailing Line/Ship Length in Deep Water, $F_d = 0.52$, Ship 42

Nondimensional H_{max} versus Distance from the Sailing Line/Ship Length in Deep Water, $F_d = 0.13$, Ship 23

Nondimensional H_{max} versus Distance from the Sailing Line/Ship Length in Shallow Water, $F_d = 0.70$, Ship 26



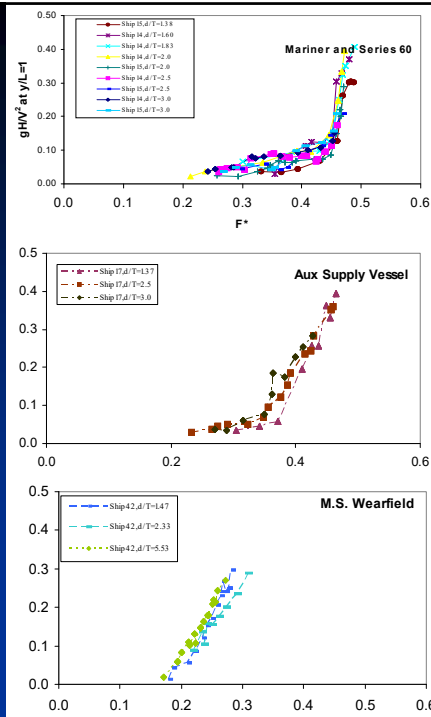
Define Modified Froude Number F^*

An empirical “Froude number” seems to collapse data from a given ship:

$$F^* = F_L \exp\left(\alpha \frac{D}{d}\right)$$

Single empirical coefficient α is dependent on ship hull form

α large for slender hulls
 α small for blocky hulls



Variation of α with Hull Form

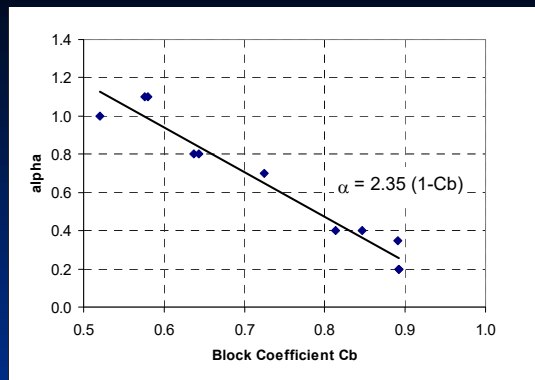
14

- Investigated dependence of α on hull form
- Seems to depend mainly on Block Coefficient

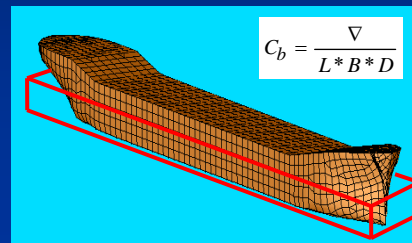
$$F^* = F_L \exp\left(\alpha \frac{D}{d}\right)$$

with

$$\alpha = 2.35(1 - C_b)$$



- General trends:
 - Streamlined hulls have α of 1 or more
 - Blunt hulls have α of 0.2 to 0.4



15

Search for Functional Relationship between gH/V^2 and F_*

- Data for a given ship shows gH/V^2 increases with F_*
 - No waves measured for F_* below 0.1
 - Data shows quadratic or higher order relationship
 - Distinct variations observed as function of hull form
- No simple mathematical function seems to ideally describe data
- Preliminary work uses quadratic expression in the form

$$\frac{gH}{V^2} = \beta (F_* - 0.1)^2$$

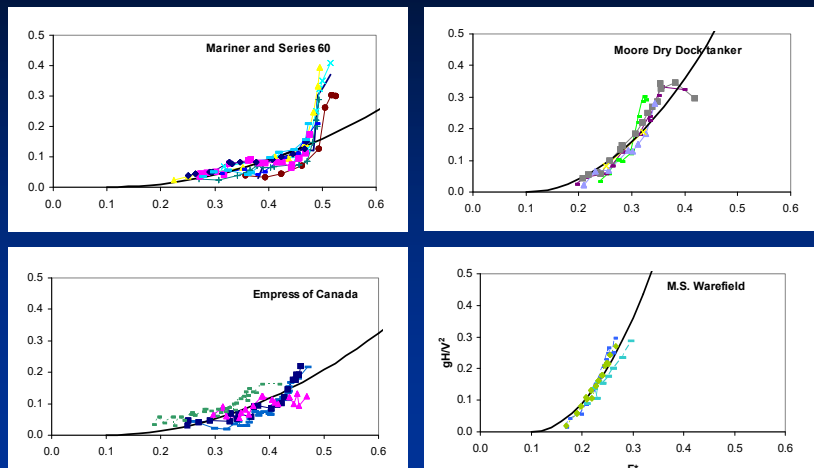
β varies with hull form and found by best fit of data for each ship

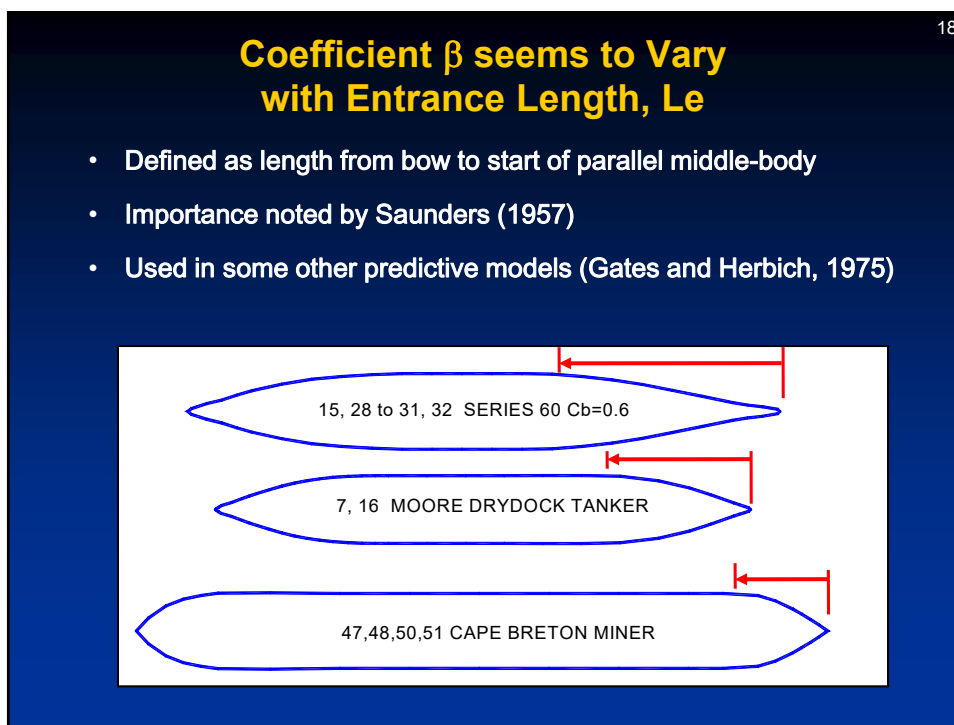
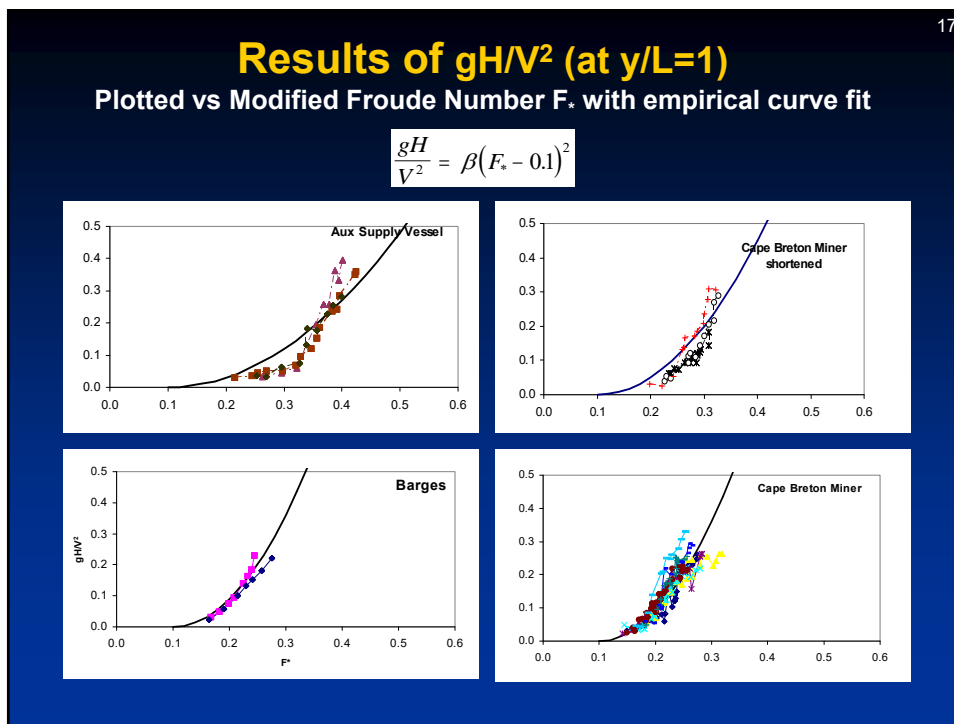
16

Results of gH/V^2 (at $y/L=1$)

Plotted vs Modified Froude Number F_* with empirical curve fit

$$\frac{gH}{V^2} = \beta (F_* - 0.1)^2$$





19

Variation of β with Hull Form

β correlated to entrance length

- Best correlation was based on L/L_e ratio
 - Streamlined ships have β of 1 to 2
 - Blunt hulls have β up to 9
- Tentative relationship:

$$\beta = 1 + 8 * \tanh^3 \left(0.45 \left(\frac{L}{L_e} - 2 \right) \right)$$

20

Evaluation of Wave Height Models

Model developed in present study compared to 1200+ data points

$$\frac{gH}{V^2} = \beta (F_* - 0.1)^2 \left(\frac{y}{L} \right)^{-1/3}$$

where

$$F_* = F_L \exp \left(\alpha \frac{T}{d} \right)$$

$$\alpha = 2.5(1 - C_b)$$

$$\beta = 1 + 8 \tanh^3 \left(0.45 \left(\frac{L}{L_e} - 2 \right) \right)$$

21

Evaluation of Wave Height Models

- Sorensen & Weggel (1984) and Weggel & Sorensen (1986)

$$H^* = \alpha (Y^*)^n$$

where

$$\log \alpha = a + b \log d^* + c (\log d^*)^2$$

with

$$a = -0.6 F_d^{-1} \quad b = 0.75 F_d^{-0.125} \quad c = 2.653 F_d - 1.95$$

and

$$n = \beta (d^*)^\delta$$

with

$$\beta = -0.225 F_d^{-0.699} \quad \text{for } 0.2 < F_d < 0.55$$

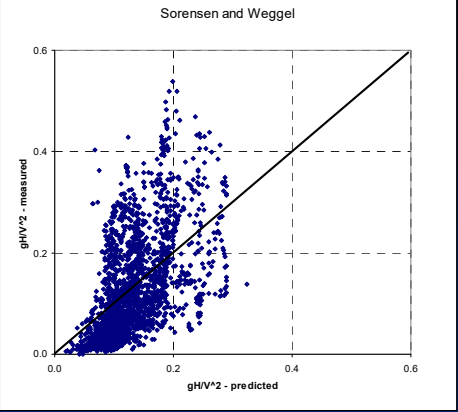
$$= -0.342 \quad \text{for } 0.5 < F_d < 0.80$$

$$\delta = -0.118 F_d^{-0.356} \quad \text{for } 0.2 < F_d < 0.55$$

$$= -0.146 \quad \text{for } 0.5 < F_d < 0.80$$

and

$$H^* = \frac{H}{\nabla^{0.33}} \quad Y^* = \frac{Y}{\nabla^{0.33}} \quad d^* = \frac{d}{\nabla^{0.33}}$$



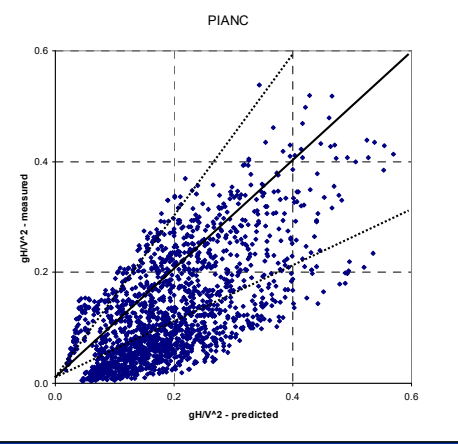
- A little complicated and removed from the physics
- Not dependent on hull form

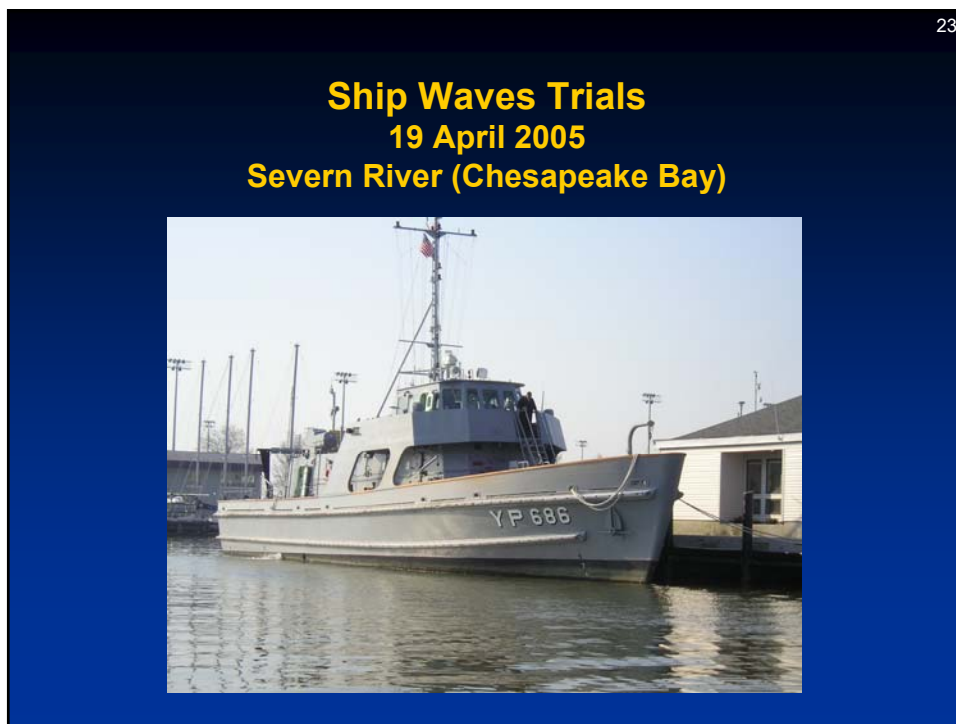
22

Evaluation of Wave Height Models

- PIANC (1987)
 - Based on work at Delft by Blaauw et al (1984) for ships in shallow confined canals

$$\frac{gH}{V^2} = F_d^2 \left(\frac{y - B/2}{d} \right)^{-1/3}$$







**YP686 approaching wave gage
(gage located 100 ft off sailing line)**

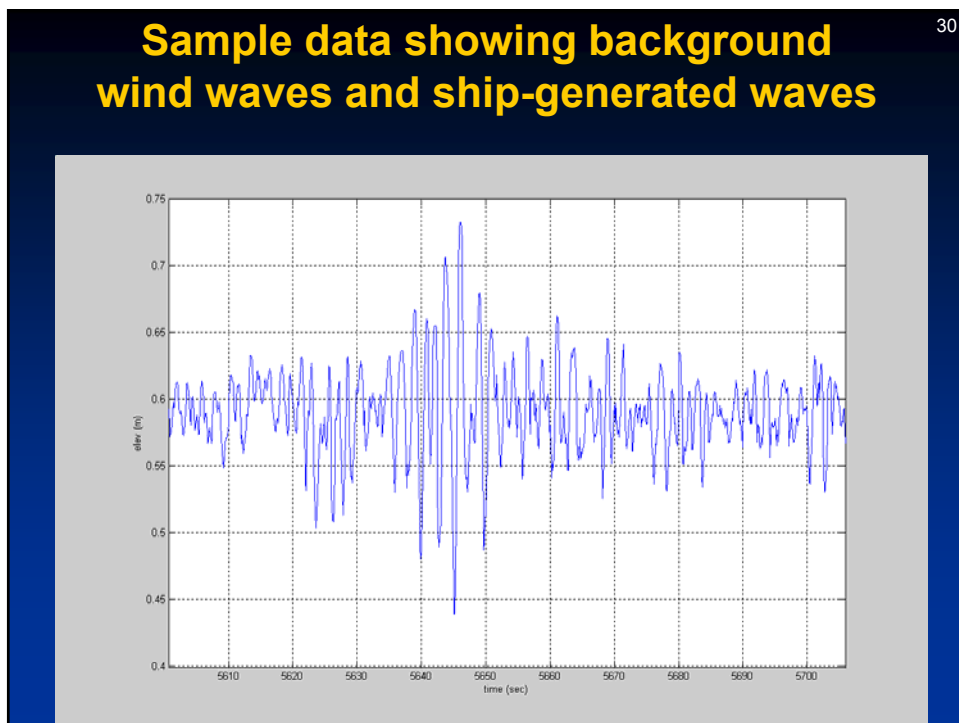
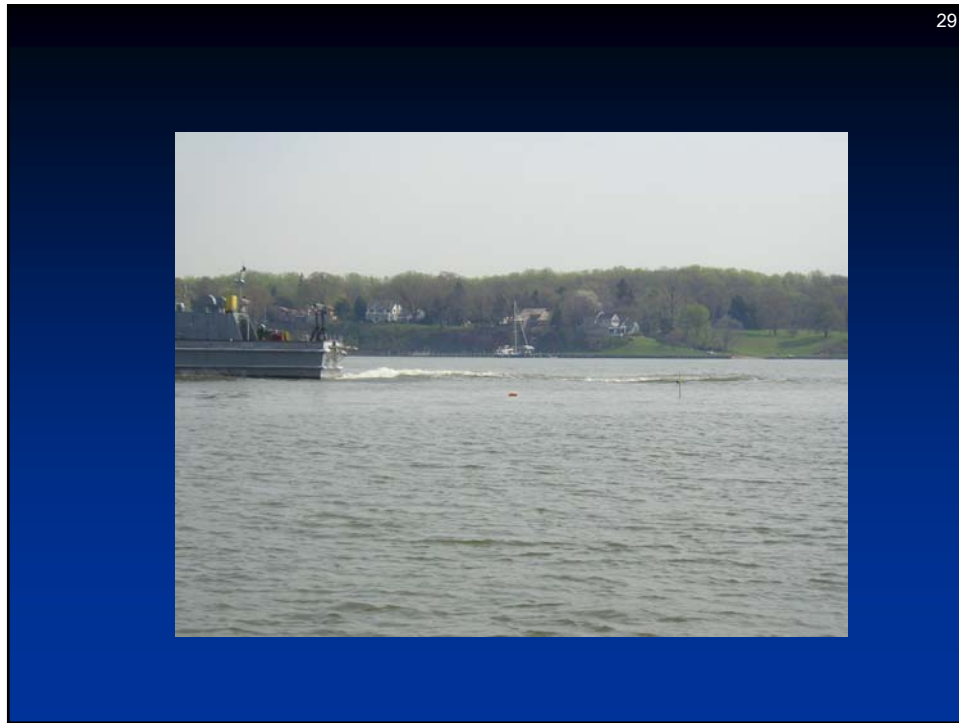
27

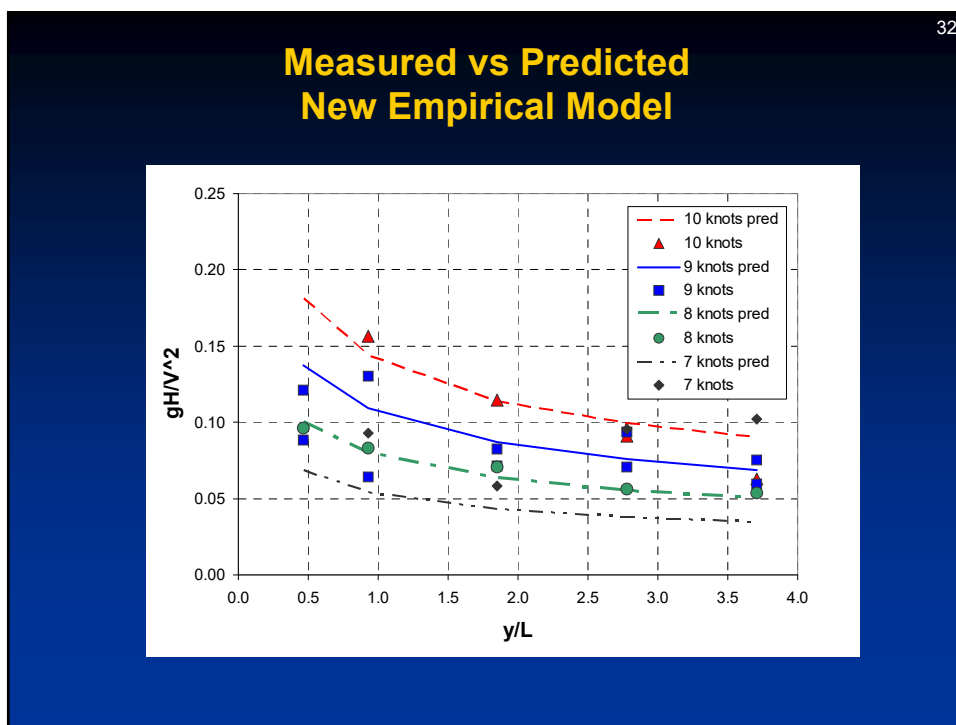
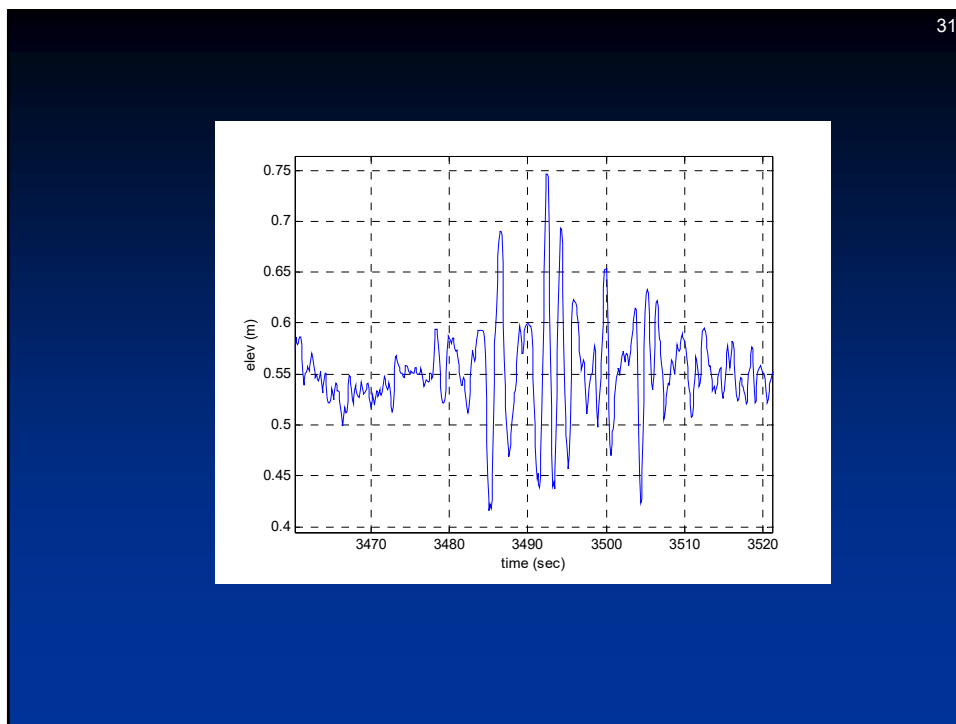


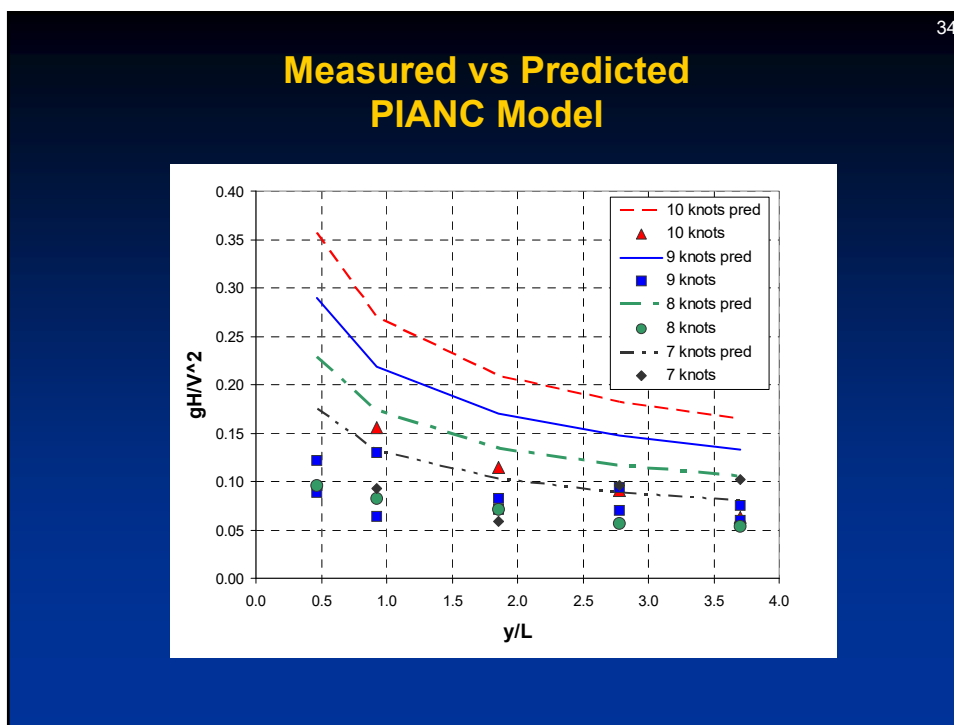
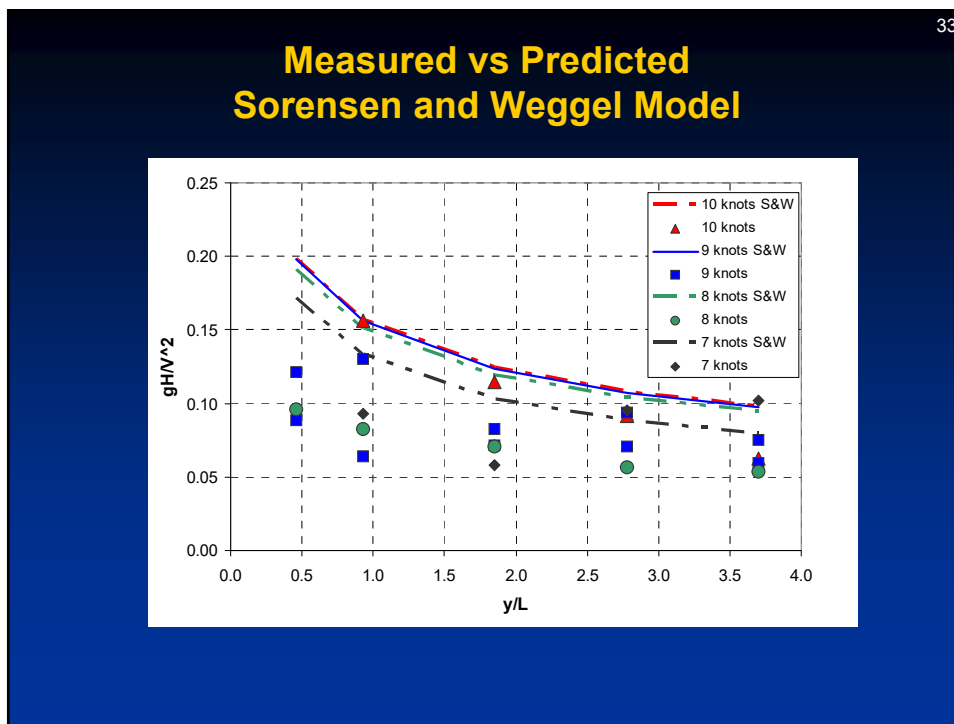
**YP686 passing wave gage
(gage located 50 ft off sailing line)**

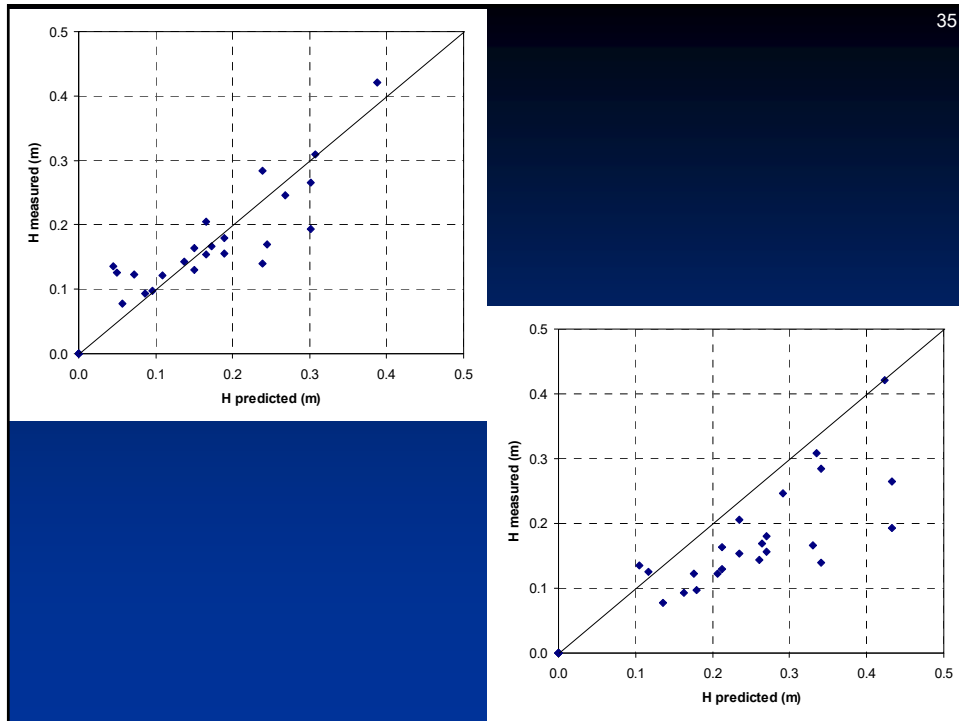
28











Summary and Conclusions Ship-Generated waves

36

- **New model gives improved predictions**
 - $\exp(\alpha T/d)$ term “unifies” data collected in various water depths
- **Model can be further improved**
 - $(y/L)^{-1/3}$ can be optimized
 - Exponent may depend on hull form
 - $(F-0.1)^2$ can be optimized
 - Higher power needed for some hull forms
 - Coefficients α and β can be improved with data from more ships
- **Need data in shallow water**
 - Lab data for very shallow water $T/d < 1.3$
 - Field data

$$\frac{gH}{V^2} = \beta (F_* - 0.1)^2 \left(\frac{y}{L} \right)^{-1/3}$$

where

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$$\alpha = 2.35(1 - C_b)$$

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